2018 IECC Commercial Mechanical Requirements
Why Care About IECC?

- Energy codes and standards set minimum efficiency requirements for new and renovated buildings, assuring reductions in energy use and emissions over the life of the building. Energy codes are a subset of building codes, which establish baseline requirements and govern building construction.

- Code buildings are more comfortable and cost-effective to operate, assuring energy, economic and environmental benefits.
Commercial Compliance Options

1. ASHRAE 90.1-2016

OR

2. 2018 IECC - Prescriptive
   - C402 - Envelope
   - C403 - Mechanical
   - C404 - SWH
   - C405 - Lighting
   AND
   - Pick At Least One C406:
     - C406.2 – Eff. HVAC Performance
     - C406.3 – Reduced Lighting Power
     - C406.4 – Enhanced Lighting Controls
     - C406.5 – On-site Supply of Renewable energy
     - C406.6 – Dedicated Outdoor Air System
     - C406.7 – High Eff. Service Water Heating
     - C406.8 – Enhanced Envelope Performance
     - C406.9 – Reduced Air Infiltration

OR

3. 2018 IECC - Performance
   - C407 – Total Building Performance
   - C402.5 – Air Leakage
   - C403.2 – Provisions applicable to all mechanical systems
   - C404 - SWH
   - Lighting Mandatory Sections
     - C405.2
     - C405.3
     - C405.4
     - C405.6
   - Building energy cost to be ≤ 85% of standard reference design building

BUILDING ENERGY CODES PROGRAM

www.energycodes.gov
• One additional efficiency feature must be selected to comply with the IECC
  – More efficient HVAC performance, OR
  – Reduced lighting power density system, OR
  – Enhanced lighting controls, OR
  – On-site supply of renewable energy
  – Dedicated outdoor air system, OR
  – More efficient SWH
Does My Project Need to Comply with the IECC?

All Buildings Other Than:

- One- and two-family residential
- R-2, R-3, R-4 three stories or less in height
Codes and standards listed in Chapter 6 are considered part of the requirements of this code to the “prescribed extent of each such reference and as further regulated in Sections C107.1.1 and C107.1.2”

- Conflicts, C107.1.1 – where differences occur between this code and the referenced codes and standards, provisions of this code apply
- Provisions in reference codes and standards, C107.1.2 – “where the extent of the reference to a referenced code or standard includes subject matter that is within the scope of this code, the provisions of this code, as applicable, shall take precedence over the provisions in the referenced code or standard”
Climate Zones - 2018 IECC

All of Alaska in Zone 7 except for the following
Boroughs in Zone 8:
- Bethel
- Dillingham
- Fairbanks N. Star
- Nome
- North Slope

Northwest Arctic
Southeast Fairbanks
Wade Hampton
Yukon–Koyukuk

Zone 1 includes
Hawaii, Guam,
Puerto Rico,
and the Virgin Islands

Warm–Humid Below White Line
Systems and equipment serving the building heating, cooling, and ventilation needs to comply with C403.2 and Sections C403.3 and C403.4 based on the equipment and systems provided.

Walk-in coolers, walk-in freezers, refrigerated warehouse coolers and refrigerated warehouse freezers shall comply with Section 403.10.1 or 403.10.2.
Single Zone Systems

Simple System
Section C403 Reorganization

- For 2018 IECC, there was a major reorganization of the mechanical section (C403)
- Rather than separate mandatory (C403.2) and prescriptive requirements by section group:
  - Similar requirements were brought together
  - Mandatory requirements were indicated (Mandatory) for each section. Sections without the “Mandatory” designation are prescriptive.
- As a result, familiar section numbers have likely changed

Mechanical sections:
- C403.1: General (Loads)
- C403.2: System Design
- C403.3: Equipment Efficiencies & Specs
- C403.4: HVAC Controls
- C403.5: Economizers
- C403.6: Multi-zone/VAV
- C403.7: Vent & Exhaust
- C403.8: Fan Eff. & Cntrl.
- C403.9: Heat Rejection
- C403.10: Refrigeration
- C403.11: Construction
- C403.12: Outside Bldg.
Mandatory Provisions
Now Spread through many Sections

Provisions Applicable to ALL Mechanical Systems

- HVAC Load Calculations
- Equipment and System Sizing
- HVAC Equipment Performance Requirements
- HVAC System Controls
- Guestroom HVAC Controls
- Hot Water Boiler Outdoor Temp. Set-back Control
- Ventilation & Vent. Control
- Energy Recovery Ventilation Systems
- Kitchen Exhaust Systems
- Duct and Plenum Insulation and Sealing
- Piping Insulation
- HVAC System Commissioning and Completion
- Air System Design and Control
- Heating Outside a Building
- Refrigeration Equipment Performance
- Walk-in Coolers and Freezers, Refrigerated Warehouse Coolers and Freezers
- Site-built walk-in Coolers and Walk-in Freezers
Heating and cooling load sizing calculations required

- ASHRAE/ACCA Standard 183 OR
- Other approved computation procedures – defined in Chapter 3
  - Interior design conditions
    - Specified by Section C302 of the IECC
      - $\leq 72^\circ F$ for heating load
      - $\geq 75^\circ F$ for cooling load

- Loads reduced from energy recovery systems utilized in the HVAC system shall be accounted for in accordance with the ASHRAE HVAC Systems and Equipment Handbook
Divided into isolation areas:

- HVAC systems serving zones > 25,000 ft² in floor area OR
- Span > one floor and are designed to operate or be occupied nonsimultaneously

Isolation areas:

- Isolation devices and controls configured to automatically shut off the supply of conditioned air and outdoor air to and exhaust air from the isolation area
- Controlled independently by a device meeting C403.4.2.2
- Central systems and plants provided with controls and devices that will allow system and equipment operation for any length of time while serving the smallest isolation area served by system or plant
Exceptions:

- Exhaust air and outdoor air connections to isolation areas where the fan system to which they connect is < 5,000 cfm
- Exhaust airflow from a single isolation area of < 10% of the design airflow of the exhaust system to which it connects
- Isolation areas intended to operate continuously or intended to be inoperative only when all other isolation areas in a zone are inoperative
• Natural and mechanical ventilation to be provided in accordance with Chapter 4 of the IMC
  – If mechanical – system to provide the capability to reduce outdoor air supply to minimum required by IMC Chapter 4
Output capacity of heating and cooling equipment only SHALL NOT be greater than calculated loads

✔ Select the system which serves the greater load, heating or cooling

- **Exceptions:**
  - Standby Equipment with Required Controls
  - Multiple Units with Combined Capacities Exceeding Loads
    - Sequencing Controls Required
Applies to all equipment used in heating and cooling of buildings

✓ Where components from different manufacturers are used
  • calculations and supporting data demonstrating combined efficiency meets requirements

Must comply with all listed efficiencies
<table>
<thead>
<tr>
<th>EQUIPMENT TYPE</th>
<th>SIZE CATEGORY</th>
<th>HEATING SECTION TYPE</th>
<th>SUBCATEGORY OR RATING CONDITION</th>
<th>MINIMUM EFFICIENCY</th>
<th>TEST PROCEDURE</th>
</tr>
</thead>
<tbody>
<tr>
<td>Air cooled (cooling mode)</td>
<td>&lt; 65,000 Btu/h</td>
<td>All</td>
<td>Split system</td>
<td>14.0 SEER</td>
<td></td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td>Single packaged</td>
<td>14.0 SEER</td>
<td>AHRI 210/240</td>
</tr>
<tr>
<td>Through-the-wall air cooled</td>
<td>≤ 30,000 Btu/h</td>
<td>All</td>
<td>Split system</td>
<td>12.0 SEER</td>
<td></td>
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<tr>
<td></td>
<td></td>
<td></td>
<td>Single packaged</td>
<td>12.0 SEER</td>
<td></td>
</tr>
<tr>
<td>Single-duct high-velocity air cooled</td>
<td>&lt; 65,000 Btu/h</td>
<td>All</td>
<td>Split system</td>
<td>11.0 SEER</td>
<td></td>
</tr>
</tbody>
</table>

(partial table; similar to other efficiency tables not shown)
• Equipment not designed for operation at AHRI Standard 550/590 test conditions of 44°F leaving chilled-water temperature and 2.4 gpm/ton evaporator fluid flow and 85°F entering condenser water temperature with 3 gpm/ton condenser water flow
  – To have maximum full-load kW/ton and NPLV ratings adjusted using Equations 4-6 and 4-7
• The FLadj and PLVadj values are only applicable for centrifugal chillers meeting all of these full-load design ranges
  – Evaporator leaving temperature ≥ 36°F
  – Condenser leaving temperature ≤ 115°F
  – 20°F ≤ LIFT ≤ 80°F
• Equipment with a leaving fluid temperature > 32°F and water-cooled positive displacement chilling packages with a condenser leaving fluid temperature < 115°F to meet Table C403.3.2(7)
  – when tested or certified with water at standard rating conditions, in accordance with the referenced test procedure
Cooling systems can’t use unless system designed with
✓ multiple steps of unloading OR
✓ Continuous capacity modulation

Capacity limited per Table C403.3.3 as limited by Section C403.5.1

<table>
<thead>
<tr>
<th>Rated Capacity</th>
<th>Maximum Hot Gas Bypass Capacity (% of total capacity)</th>
</tr>
</thead>
<tbody>
<tr>
<td>≤ 240,000 Btu/h</td>
<td>50%</td>
</tr>
<tr>
<td>&gt; 240,000 Btu/h</td>
<td>25%</td>
</tr>
</tbody>
</table>
Boiler systems with design input $\geq 1,000,000$ Btu/h to comply with turndown ratio specified in Table C403.3.4

<table>
<thead>
<tr>
<th>BOILER SYSTEM DESIGN INPUT (Btu/h)</th>
<th>MINIMUM TURNDOWN RATIO</th>
</tr>
</thead>
<tbody>
<tr>
<td>$\geq 1,000,000$ and less than or equal to $5,000,000$</td>
<td>3 to 1</td>
</tr>
<tr>
<td>$&gt; 5,000,000$ and less than or equal to $10,000,000$</td>
<td>4 to 1</td>
</tr>
<tr>
<td>$&gt; 10,000,000$</td>
<td>5 to 1</td>
</tr>
</tbody>
</table>

System turndown requirement must be met through the use of

- multiple single input boilers OR
- $\geq 1$ modulating boilers OR
- combination of single input and modulating boilers
Thermostatic Controls (Mandatory)
Section C403.4.1

Control required for each system
✓ if zoned for each zone

Exceptions: Independent perimeter systems that are designed to offset only building envelope heat losses or gains or both serving one or more perimeter zones also served by an interior system provided:

1. The perimeter system includes at least one thermostatic control zone for each building exposure having exterior walls facing only one orientation (within +/- 45 degrees) (0.8 rad) for more than 50 contiguous feet (15.2 m); and

2. The perimeter system heating and cooling supply is controlled by a thermostat(s) located within the zone(s) served by the system.
Heat pump systems

- Heat pump thermostat required when supplying electric resistance heating
- Control must prevent supplemental heat demand when heat pump can meet the heating load.
  - Except during defrost
- Can be met by outside air temperature lockout set to
  - lock out resistance heat when the heat pump can meet load
  - Typically around 35°F
Thermostats must have at least a 5°F deadband between operation of heating and cooling

✓ **Exception:**
  - Thermostats requiring manual change over between heating and cooling
  - Occupancies or applications requiring precision in indoor temperature control as approved by code official
• Where separate thermostatic control devices for heating and cooling in a zone:
  – Limit switch
  – Mechanical stop OR
  – Direct digital control system with software programming shall be **configured** to prevent heating set point from exceeding cooling set point to maintain deadband
Heated or Cooled Vestibules (Mandatory)
Section C403.4.1.4

- Heating system for heated vestibules and air curtains with integral heating
  - Controls configured to shut off heat when outdoor air temperature is > 45°F
- Heating and cooling systems controlled by thermostat in vestibule configured to limit heating to < 60°F and cooling to > 85°F

**Exception:** control of heating or cooling provided by
- site-recovered energy or
- transfer air that would otherwise be exhausted
Hot water boilers that supply heat to the building through one or two-pipe heating systems to have an outdoor setback control that lowers the boiler water temperature based on the outdoor temperature.
Automatic time clock or programmable system

- **Exceptions:**
  - Zones operated continually
  - Zones with full HVAC load demand <6,800 Btu/h and has a shut off switch located with ready access

Thermostatic setback capabilities (C403.4.2.1)

- Capability to maintain zone temps down to 55ºF or up to 85ºF

Automatic setback and shutdown (C403.4.2.2)

- Seven different daily schedules/week
- Retain programming ≥ 10 hrs with loss of power **AND**
  - Manual override up to 2 hours **OR**
  - Manual timer **configured to operate** up to 2 hours **OR**
  - Occupancy sensor
Automatic start controls for each HVAC system

✓ Configured to automatically adjust daily start time to bring each space to desired occupied temperature immediately prior to scheduled occupancy
Hydronic System Controls
Section C403.4.3

✔ Limit reheat/recool of fluids
✔ Multiple boiler heating plants must include automatic controls configured to sequence operation of the boilers
✔ Single boilers > 500,000 Btu/h input design capacity must include multi-staged or modulating burner
3-Pipe System – not allowed (C403.4.3.1)
  ✓ Can’t use a common return

2-Pipe Changeover System (C403.4.3.2)
  ✓ Dead band between changeover ≥ 15°F outside temperature

Diagram Courtesy of Ken Baker
Temperature dead band **configured to at least 20°F** *(C403.4.2.3.1)*

**Exception:**
- Where system loop temp optimization controller is installed and can determine the most efficient operating temp based on real time conditions of demand and capacity

Example:
- Heat rejection off below 75°F loop temperature.
- Boiler off above 55°F loop temperature
  
  \[
  75°F - 55°F = 20°F \text{ dead band}
  \]
Heat rejection equipment in Climate Zones 3 - 8

✓ Closed-circuit cooling tower used directly in heat pump loop
  • Install either automatic valve to bypass flow of water around tower, **except for minimal freeze protection flow** OR lower leakage positive closure dampers to be provided

✓ Open-circuit tower used directly in heat pump loop
  • Install automatic valve to bypass all heat pump water flow around tower

✓ Open- or closed-circuit tower used in conjunction with separate heat exchanger to isolate cooling tower from heat pump loop
  • Heat loss controlled by shutting down the circulation pump on cooling tower loop

Exception: Where it can be demonstrated that a heat pump system will be required to reject heat throughout the year

*Note: requirement allowing only the heat exchanger option in climate zones 5-8 has been removed; all three options can be used in 3-8 now.*
Two position shutoff valve

- Required on each hydronic heat pump where total system pump power > 10 hp
System ≥ 300,000 Btu/h heating or cooling must include

- Temperature reset and variable flow
  - Configured automatic resets for supply water temperature by at least 25% of design supply-to-return temperature differences and
  - Automatic vary fluid flow if a combined motor capacity ≥ 2 hp with ≥ 3 control valves or other devices:
    - Must reduce system design flow rate by ≥ 50% by designed valves or pumps that modulate flow:
      - Modulating valves or VSD on pumps,
      - Valves that step open and close OR
      - Sequencing pumps or valves turn on and of as function of load
Hydronic System Part Load Control
Section C403.4.4 – Cont’d

• Automatic vary pump flow on heating-water systems, chilled-water systems and heat rejection loops serving cooled unitary air conditioners
  – Where pumps operate continuously or based on a time schedule, pumps with nominal output hp of \( \geq 2 \text{hp} \) to have a variable speed drive
  – Where pumps have automatic direct digital control configured to operate pumps only when zone heating or cooling is required, a variable speed drive to be provided for pumps with motors having the same or greater nominal output power per Table C403.4.4 based on climate zone and system served

• Where a variable speed drive is required per above bullet, pump motor power input to be not more than 30% of design wattage at 50% of design water flow. Pump flow rate to be controlled to maintain one control valve nearly wide open or to satisfy the minimum differential pressure.
 Exceptions:

- Supply-water temp. reset for chilled-water systems supplied by off-site district chilled water or chilled water from ice storage systems

- Variable pump flow not required on
  - dedicated coil circulation pumps for freeze protection
  - dedicated equipment circulation pumps where configured in primary/secondary design to provide the minimum flow requirements of the equipment manufacturer for proper operation of equipment

- Variable speed drives not required on heating water pumps where more than 50% of annual heat is generated by electric boiler
Multiple chiller chilled water plants

- Capable and configured to reduce flow through the chiller automatically when chiller is shut down
- Chillers piped in series considered one chiller

Multiple boiler plants

- Capable and configured to reduce flow through the boiler system automatically when boiler is shut down

What to look for:
Separate pumps for each boiler or chiller with check valves, OR
A variable flow pump with isolation valves for each boiler or chiller
### Economizers

**Section C403.5**

<table>
<thead>
<tr>
<th>CLIMATE ZONES</th>
<th>ECONOMIZER REQUIREMENT</th>
</tr>
</thead>
<tbody>
<tr>
<td>1A</td>
<td>No economizer requirement</td>
</tr>
</tbody>
</table>
| 2A, 2B, 3A, 3B, 3C, 4A, 4B, 4C, 5A, 5B, 5C, 6A, 6B, 7, 8 | Economizers on individual DX cooling units ≥ 54,000 Btu/h<sup>a</sup>  
Except for climate zone 1B  
Economizer (usually central water economizer) on any chilled water cooling unit if total cooling meets table C403.3 limits |

Total supply capacity of all fan-cooling units not provided with economizers shall not exceed 20% of the total supply capacity of all fan-cooling units in the building or 1,500,000 Btu/h, whichever is greater, unless otherwise excepted.
## Economizers – Table C403.5(1)

**Section C403.5**

<table>
<thead>
<tr>
<th>Climate Zones (Cooling)</th>
<th>Total Chilled-Water System Capacity &lt; Capacity of Cooling Units with Air Economizers</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>Local Water-cooled Chilled-water Systems</td>
</tr>
<tr>
<td>1A</td>
<td>No economizer requirement</td>
</tr>
<tr>
<td>1B, 2A, 2B</td>
<td>960,000 Btu/h</td>
</tr>
<tr>
<td>3A, 3B, 3C, 4A, 4B, 4C</td>
<td>720,000 Btu/h</td>
</tr>
<tr>
<td>5A, 5B, 5C, 6A, 6B, 7, 8</td>
<td>1,320,000 Btu/h</td>
</tr>
</tbody>
</table>

Example: Hotel with guest room chilled water fan coil units totaling 1,500 MBH of cooling capacity in climate zone 5A
- Central water economizer or individual fan coil air economizers required if a water-cooled chiller
- No economizer requirement if all air-cooled chillers or district chilled water source outside of building
Exceptions: (economizers not required)

- Individual fan systems not served by chilled water in Climate Zones 1A and 1B
- Where > 25% of air designed to be supplied by the system is to spaces that are designed to be humidified > 35°F dew-point temperature to satisfy process needs
- Systems expected to operate < 20 hours/week
- Systems serving supermarket areas with open refrigerated casework systems
- Where cooling efficiency meets or exceeds efficiency requirements in Table C403.5(2)
- Systems that include a heat recovery system in accordance with Section C403.9.5
Trade-off high cooling efficiency for economizer

<table>
<thead>
<tr>
<th>CLIMATE ZONES</th>
<th>COOLING EQUIPMENT PERFORMANCE IMPROVEMENT (EER OR IPLV)</th>
</tr>
</thead>
<tbody>
<tr>
<td>2B</td>
<td>10% Efficiency Improvement</td>
</tr>
<tr>
<td>3B</td>
<td>15% Efficiency Improvement</td>
</tr>
<tr>
<td>4B</td>
<td>20% Efficiency Improvement</td>
</tr>
</tbody>
</table>
• Systems to be integrated with the mechanical cooling system and **be configured to provide** partial cooling even where additional mechanical cooling is required to provide the remainder of the cooling load

• Controls shall not be capable of creating a false load in the mechanical cooling systems by limiting or disabling the economizer or any other means, such as hot gas bypass, except at the lowest stage of mechanical cooling
Units that include an air economizer to comply with the following:

- Unit controls are configured so the mechanical cooling capacity control is interlocked with the air economizer controls such that the outdoor air damper is at the 100% open position when mechanical cooling is on and the outdoor air damper does not begin to close to prevent coil freezing due to minimum compressor run time until the leaving air temp. < 45°F.

- Direct expansion (DX) units that control ≥75,000 Btu/h of rated capacity of the capacity of the mechanical cooling directly based on occupied space temp. shall have not fewer than two stages of mechanical cooling capacity.

- Other DX units including those that control space temp. by modulating the airflow to the space be in accordance with Table C403.5.1.
HVAC system design and economizer controls shall be such that economizer operation does not increase building heating energy use during normal operations

**Exception:** Economizers on variable air volume (VAV) systems that cause zone level heating to increase due to a reduction in supply air temperature
Configured to modulate outdoor air and return air dampers to provide up to 100% of design supply air quantity as outdoor air for cooling
Economizer dampers to be configured to sequence with mechanical cooling equipment and not be controlled by only mixed air temperature

**Exception:**
- Can use mixed air temperature limit control for systems controlled from space temperature
  - Example: single-zone systems
Air economizers to be configured to automatically reduce outdoor air intake to design minimum outdoor air quantity when outdoor air intake will no longer reduce cooling energy usage

High-limit shutoff control types to be chosen from Table C403.5.3.3 for specific climates

Specifications for high-limit shutoff control type settings per Table C403.5.3.3
Air Economizers – Relief of Excess Outdoor Air

Section C403.5.3.4

✔ Systems to be capable of relieving excess outdoor air during air economizer operation to prevent over-pressurizing the building

✔ Relief air outlet to be located to avoid recirculation into the building
Return, exhaust/relief and outdoor air dampers used in economizers shall comply with Section C403.7.7 (Shut Off Dampers)
Configured to cool supply air by indirect evaporation and provide up to 100% of expected system cooling load at outdoor air temperatures < 50°F dry bulb/45°F wet bulb

**Exceptions:**
- Systems primarily serving computer rooms in which 100% of expected system cooling load at 40°F dry bulb/35°F wet bulb is met with evaporative water economizers
- Systems primarily serving computer rooms with dry cooler water economizers which satisfy 100% of the expected system cooling load at 35°F dry bulb
- Systems where dehumidification requirements cannot be met using outdoor air temps of 50°F dry bulb/45°F web bulb and where 100% of expected system cooling load at 45°F dry bulb/40°F wet bulb is met with evaporative water economizers
Precooling coil and water-to-water heat exchangers used as part of a water economizer system to have either a:

- water side pressure drop < 15 feet of water **OR**
- secondary loop created so that the coil or heat exchanger pressure drop is not seen by the circulating pumps when the system is in normal cooling (non-economizer) mode
• Air cooled unitary direct-expansion units (listed in Tables C403.3.2(1-3) and variable refrigerant flow (VRF) units that are equipped with an economizer per C403.5 to C403.5.4 to include a fault detection and diagnostics (FDD) system complying with the following:
  – Temperature sensors permanently installed to monitor system operation
    • Outside air
    • Supply air
    • Return air
  – Temperature sensors have an accuracy of $\pm 2^\circ F$ over the range of 40°F - 80°F
  – Refrigerant pressure sensors, where used, have an accuracy of $\pm 3\%$ of full scale
• Unit controller configured to provide system status by indicated the following:
  – Free cooling available
  – Economizer enabled
  – Compressor enabled
  – Heating enabled
  – Mixed air low limit cycle active
  – Current value of each sensor

• Unit controller capable of manually initiating each operating mode so that the operation of compressors, economizers, fans and heating system can be independently tested and verified
Unit configured to report faults to a fault management application accessible by day-to-day operating or service personnel, or annunciated locally on zone thermostats.

The FDD system configured to detect the following faults:

- Air temperature sensor failure/fault
- Not economizing when the unit should be economizing
- Economizing when the unit should not be economizing
- Damper not modulating
- Excess outdoor air
• Supply air systems serving multiple-zones shall be VAV systems with zone controls to reduce reheated air to one of the following:
  – 20% of zone peak supply for DDC (30% others)
  – DDC with 20% air in deadband, first stage heating at 20%, second stage heating up to 50% zone airflow
  – Airflow required to meet IMC ventilation requirements
  – A higher rate shown to use less energy through system OA reduction
  – Airflow required for accreditation standards

• Exceptions:
  – Zones or supply air systems where not less than 75% of the energy for reheating or for providing warm air in mixing systems is provided from a site-recovered energy source.
  – Systems that prevent reheating, recooling, mixing or simultaneous supply of air that has been previously cooled (mechanically or economizer) and heated air.
Single duct VAV systems to use terminal devices capable of and configured to reduce the supply of primary supply air before reheating or recooling takes place
Systems with one warm air duct and one cool air duct to use terminal devices configured to reduce flow from one duct to a minimum before mixing of air from the other duct takes place.
Individual dual duct or mixing reheating and cooling systems with a single fan and with total capacities > 90,000 Btu/h (7.5 tons) should not have economizers.
Supply-Air Temperature Reset Controls  
Section C403.6.5

Multiple zone HVAC systems to have controls to automatically reset supply-air temperature in response to building loads or outdoor air temperature.

Controls to be **configured to reset** supply air temperature at least 25% of difference between design supply-air temperature and design room air temperature.

**Exceptions:**

- Systems that prevent reheating, recooling or mixing of heated and cooled supply air
- 75% of energy for reheating is from site-recovered or site solar energy sources
- Zones with peak supply air quantities of ≤ 300 cfm
Multiple-zone VAV systems with direct digital control of individual zone boxes reporting to central control panel

- Automatic controls configured to reduce outdoor air intake flow below design rates in response to changes in system ventilation efficiency (Ev) as defined by IMC

**Exceptions:**

- VAV systems with zonal transfer fans that recirculate air from other zones without directly mixing it with outdoor air, dual-duct dual-fan VAV systems, and VAV systems with fan-powered terminal units
- Systems where total design exhaust airflow is > 70% of total design outdoor air intake flow requirements
Parallel fan-powered VAV zone control

Section C403.6.7

• Parallel fan-powered VAV zone terminal units shall be configured to operate as follows:
  – Turn off the terminal fan except when space heating is required or where required for ventilation.
  – Turn on the terminal fan as the first stage of heating before the heating coil is activated.
  – During heating for warmup or setback temperature control, either:
    • Operate the terminal fan and heating coil without primary air.
    • Reverse the terminal damper logic and provide heating from the central air handler by primary air.
• Systems with direct digital control of individual reporting to the central control panel
  – Static pressure set point to be reset based on the zone requiring the most pressure (i.e., the set point is reset lower until one zone damper is nearly wide open)
  – Direct digital controls capable of monitoring zone damper positions or have an alternative method of indication the need for static pressure **configured to provide** all of the following:
    • Automatically detecting any zone that excessively drives the reset logic
    • Generating an alarm to the system operational location
    • Allowing an operator to readily remove > 1 zones from the reset algorithm
Static Pressure Sensor Location
Section C403.6.9

- Sensors used to control VAV fans
  - Placed so that the controller setpoint is ≤ 1.2 inches w.c.

- Sensors installed downstream of major duct splits
  - At least one sensor to be located on each major branch so that static pressure can be maintained in each branch
DCV must be provided for each zone with spaces > 500 ft² and the average occupant load ≥ 25 people/1000 ft² of floor area (per IMC table 403.3.1.1) where the HVAC system has:

- An air-side economizer, or
- Automatic modulating control of the outdoor air damper, or
- A design outdoor airflow > 3,000 cfm

Demand control ventilation (DCV): a ventilation system capability that provides for the automatic reduction of outdoor air intake below design rates when the actual occupancy of spaces served by the system is less than design occupancy.
Exceptions:

- Systems with energy recovery per C403.7.4
- Multiple zone systems without direct digital control of single zones communicating with central control panel
- Systems with design outdoor airflow < 1,200 cfm
- Spaces where supply airflow rate minus any makeup or outgoing transfer air requirement < 1,200 cfm
- Ventilation provided only for process loads
• Garages for storing or handling automobiles operating under their own power shall employ contamination-sensing devices and automatic controls configured to stage fans or modulate fan average airflow rates to <50% of design capacity, or intermittently operate fans < 20% of occupied time or as required to maintain acceptable contaminant levels in accordance with IMC provisions.

• Failure of these devices shall cause the exhaust fans to operate continuously at design airflow.

Exceptions:
- Garages with total exhaust capacity < 22,500 cfm with ventilation systems that do not utilize heating or mechanical cooling.
- Garages that have garage area to ventilation system motor nameplate power ratio >1,125 cfm/hp and do not utilize heating or mechanical cooling.
Units providing ventilation air to multiple zones and operating in conjunction with zone heating and cooling systems to not use heating or heat recovery to warm supply air to a temperature > 60°F when representative building loads or outdoor air temperatures indicate majority of zones require cooling.

Things to look for:

- Applies to DOAS systems with or without heating or cooling capability
- HRV or ERV systems will require bypass of flat plate exchangers or speed control of heat wheels
- “Neutral air” control is not allowed
Energy Recovery Ventilation Systems (Mandatory)
Section C403.7.4

- Applies to fan systems with supply airflow rates > values in Tables C403.7.4(1-2)
  - Note that prior 0 cfm values in tables have been increased
- Exhaust air total recovery efficiency must be ≥ 50%
- When an air economizer is required
  - include a bypass or controls that permit operation of economizer per C403.5

Energy recovery ventilation (ERV) systems: employ air-to-air heat exchangers to recover energy from exhaust air for the purpose of preheating, precooling, humidifying or dehumidifying outdoor ventilation air prior to supplying the air to a space, either directly or as part of an HVAC system.
Energy Recovery Ventilation Systems (Mandatory)
Section C403.7.4 – Cont’d

Exceptions:

✓ Where energy recovery ventilation systems prohibited by the IMC
✓ Lab fume hood system with at least one of the following:
  – VAV hood exhaust and room supply systems configured to reduce exhaust and makeup air volume to ≤ 50% of design values
  – Direct makeup (auxiliary) air supply equal to at least 75% of exhaust rate, heated no warmer than 2ºF below room setpoint, cooled to no cooler than 3ºF above room setpoint, with no humidification added, and no simultaneous heating and cooling use for dehumidification control
✓ Systems serving uncooled spaces and heated to < 60ºF
✓ Where > 60% of outdoor heating energy is from site-recovered or site solar energy
✓ Heating energy recovery in Climate Zones 1-2
✓ Cooling energy recovery in Climate Zones 3C, 4C, 5B, 5C, 6B, 7, and 8
✓ Systems requiring dehumidification that employ energy recovery in series with the cooling coil
✓ Where largest source of air exhausted at a single location at building exterior is < 75% of design outside air flow rate
✓ Systems expected to operate < 20 hours/week at outdoor air % covered by Table C403.7.4(1)
✓ Systems exhausting toxic, flammable, paint or corrosive fumes or dust
✓ Commercial kitchen hoods used for collecting and removing grease vapors and smoke
• Replacement air introduced directly into the exhaust hood cavity shall not be > 10% of the hood exhaust airflow rate

• Conditioned supply air delivered to any space shall not exceed the greater of the following:
  – Ventilation rate required to meet the space heating or cooling load
  – Hood exhaust flow minus the available transfer air from adjacent space where available transfer air is considered to be that portion of outdoor ventilation air not required to satisfy other exhaust needs, such as restrooms, and not required to maintain pressurization of adjacent spaces
• Total kitchen hood exhaust flow rate >5,000 cfm, each hood be a factory built commercial exhaust hood listed by nationally recognized testing laboratory in compliance with UL 710

• Each hood shall have a maximum exhaust rate as specified in Table C403.7.5

Where a single hood, or hood section, is installed over appliances with different duty ratings, the maximum allowable flow rate for the hood or hood section shall be based on the requirements for the highest appliance duty rating under the hood or hood section

• **Exception**: where at least 75% of the replacement air is transfer air that would otherwise be exhausted
• Each kitchen exhaust hood shall comply with one of the following:
  – Not < 50% of all replacement air shall be transfer air that would otherwise be exhausted
  – Demand ventilation systems on not < 75% of the exhaust air that are configured to provide not less than 50% reduction in exhaust and replacement air system airflow rates including controls necessary to modulate airflow in response to appliance operation and maintain full capture and containment of smoke, effluent and combustion products during cooking and idle
  – Listed energy recovery devices with a sensible heat recovery effectiveness not <40% on not <50% of the total exhaust airflow
• In Group R-1 buildings with > 50 guestrooms, each guestroom to be provided with controls complying with C403.7.6.1 and C403.7.6.2.

  – For these systems room occupancy sensing can be either:
    • Occupant sensors, or
    • Card key controls
  – For these systems unrented rooms can be determined either:
    • With a signal from a networked reservation, or
    • After 16 hours of continuous vacancy
Automatic Control of HVAC Systems Serving Guestrooms (Mandatory)

Section C403.7.6.1 - Temperature Setpoint Controls

- Capable of and configured to
  - Automatically raise cooling setpoint and lower heating setpoint by not less than 4°F from occupant setpoint within 30 minutes after occupants leave (vacant)
  - Automatically raise cooling setpoint to not lower than 80°F and lower heating setpoint to not higher than 60°F when guestroom is unrented and is unoccupied more than 30 minutes (unrented)

- Not precluded
  - Networked control systems capable of returning thermostat setpoints to default occupied setpoints 60 minutes prior to time a guestroom is scheduled to be occupied
  - Cooling capable of limiting relative humidity with a setpoint not lower than 65% relative humidity during unoccupied periods
Automatic Control of HVAC Systems Serving Guestrooms (Mandatory)
Section C403.7.6.2 - Ventilation Controls

- Capable of and configured to
  - Automatically turn off ventilation and exhaust fans within 30 minutes of occupants leaving OR
  - Isolation devices provided to each guestroom capable of automatically shutting off supply of outdoor air to and exhaust air from guestroom

Exception: Guestroom ventilation systems not precluded from having an automatic daily pre-occupancy purge cycle that provides daily outdoor air ventilation during unrented periods at design ventilation rate for 60 minutes, or at a rate and duration equivalent to one air change
Shutoff Dampers (Mandatory)
Section C403.7.7

- Outdoor air intake and exhaust openings and stairway and shaft vents provided with Class I motorized dampers
- Dampers with air leakage rate ≤ 4 cfm ft² of damper surface at 1.0 inch water gauge (249 Pa) and labeled and approved in accordance with AMCA 500D
- Outdoor air intake and exhaust dampers with automatic controls configured to close the systems or spaces served when not in use or during unoccupied period warm-up and setback operation
  - Unless systems served require outdoor or exhaust air per IMC
  - OR
  - Dampers are opened to provide intentional economizer cooling
• Stairway and shaft vent dampers installed with automatic controls configured to open up on activation of any fire alarm initiating device of the building’s fire alarm system or the interruption of power to the damper

  ✓ **Exceptions** non-motorized gravity dampers permitted:
  • in buildings < 3 stories
  • for buildings of any height located in Climate Zones 1-3
  • outside air intake or exhaust airflows of 300 cfm (0.14 m³/s) or less
Shutoff Dampers (Mandatory)
Section C403.7.7 – Cont’d

• Gravity motorized dampers shall have an air leakage rate:
  – \(< 20 \text{ cfm/ft}^2 \) where \( > 24 \) inches in either dimension
  – \( 40 \text{ cfm/ft}^2 \) where \( < 24 \) inches in either dimension

• Air leakage rate determined at 1.0 inch water gauge (249 Pa) when tested in accordance with AMCA 500D

• Dampers labeled by an approved agency
Applies when all fans in a system exceed a total of 5 hp motor nameplate for all fans operating at design conditions.

- Nameplate method
- Brake hp (BHP) method

BHP option includes adjustment “adders” and “deducts” for certain devices

Single-zone variable air volume systems shall comply with the constant volume fan power limitation

Table C403.8.1(2)
Fan Power Limitation Pressure Drop Adjustment

<table>
<thead>
<tr>
<th>DEVICE</th>
<th>ADJUSTMENT</th>
</tr>
</thead>
<tbody>
<tr>
<td>Return air or exhaust systems required by code or accreditation</td>
<td>0.5 in w.c. (2.15 in w.c. for laboratory and vivarium systems)</td>
</tr>
<tr>
<td>standards to be fully ducted, or systems required to maintain air</td>
<td></td>
</tr>
<tr>
<td>pressure differentials between adjacent rooms</td>
<td></td>
</tr>
<tr>
<td>Return and exhaust air flow control devices</td>
<td>0.5 in w.c.</td>
</tr>
<tr>
<td>Exhaust filters, scrubbers, or other exhaust treatment.</td>
<td>The pressure drop of device calculated at fan system design condition.</td>
</tr>
<tr>
<td>Particulate Filtration Credit: MERV 9 thru 12</td>
<td>0.5 in w.c.</td>
</tr>
</tbody>
</table>

(Partial table)
Exceptions:

- Hospital, vivarium, and laboratory systems using flow control devices on exhaust and/or return for health and safety or environmental control permitted to use variable fan power limitation.
- Individual exhaust fans with motor nameplate ≤ 1 hp.
Selected fan motor to be no larger than first available motor size greater than bhp

Fan bhp on design documents

**Exceptions**

- Fans < 6 bhp, where first available motor larger than bhp has nameplate rating within 50% of bhp, next larger nameplate motor size may be selected
  
  Example: 5.2 Bhp; next size of 7.5 is within 1.5 * Bhp (7.8) so may upsize to 10 HP, or next size after 7.5 HP

- Fans ≥ 6 bhp, where first available motor larger than bhp has nameplate rating within 30% of bhp, next larger nameplate motor size may be selected

- Fans with motor nameplate hp < 1 hp

bhp = brake horsepower
• Have a fan efficiency grade (FEG) < 67 as determined in accordance with AMCA 205 by an approved independent testing laboratory or labeled by the manufacturer
• Total efficiency at the design point of operation be within 15 percentage points of the max. total efficiency of the fan

• **Exceptions:**
  – Fans of ≤ 5 hp
    • Individual fans with a motor nameplate horsepower of ≤5 hp UNLESS
    • Multiple fans in a series or parallel that have a combined motor nameplate horsepower ≤ 5hp and are operated as the functional equivalent of a single fan
Exceptions (cont’d)

• Fans that are part of the equipment covered in Section C403.3.2

• Fans included in an equipment package certified by an approved agency for air or energy performance

• Powered wall/roof ventilators

• Fans outside the scope of AMCA 205

• Fans that are intended to operate only during emergency conditions
• Motors for fan ≥ 1/12 hp and < 1 hp shall be electronically commutated motors OR have a minimum motor efficiency of 70% rated in accordance with DOE 10 CFR 431
• Motors must have the means to adjust motor speed for either balancing or remote control
• The use of belt-driven fans with sheave adjustments for airflow balancing instead of a varying motor speed is permitted

Exceptions:
– Motors in the airstream within fan coils and terminal units that only provide heating to the space served
– Motors in space-conditioning equipment that comply with Section C403.3.2 or C403.8.1 through C403.8.3
– Motors that comply with Section C405.7
Each cooling system listed in Table C403.8.5.1 (DX ≥65 MBH capacity & chilled water or evaporative with fan ≥ ¼ hp) to be designed to vary the indoor fan airflow as a function of load and comply with the following:

- Direct expansion (DX) and chilled water cooling units that control capacity of mechanical cooling directly based on space temp to have not fewer than 2 stages of fan control
  - Low or minimum speed ≤ 66% full speed
    - Fan to draw ≤ 40% of fan power at full fan speed
    - Used during period of low cooling load and ventilation-only operation

- Other units including DX cooling and chilled water that control the space temp. by modulating the airflow to the space have modulation fan control (usually a variable speed drive)
  - Minimum speed ≤ 50% of full speed
    - Fan to draw ≤ 30% of fan power at full fan speed
  - Low or minimum speed used during period of low cooling load and ventilation-only operation
Units that include an airside economizer in accordance with Section C403.5 to have not fewer than two speeds of fan control during economizer operation

**Exceptions:**

- Modulating fan control is not required for chilled water and evaporative cooling units with fan motors < 1 hp where the units are not used to provide ventilation air and the indoor fan cycles with the load
- Where the volume of outdoor air required to comply with the ventilation requirements of IMC at low speed exceeds the air that would be delivered at the speed defined in Section C403.8.5
  - Minimum speed to be selected to provide the required ventilation air
Each tower fan powered by a motor ≥ 7.5 hp must include variable speed or two speed fan

- Have controls to automatically change the fan speed to control the leaving fluid temperature or condensing temperature/pressure of the heat rejection device

**Exception:**
- Heat rejection devices where energy usage is included in the ratings in accordance with Tables C403.3.2(6) and C403.3.2(7)
• Air-cooled condensers, dry coolers, open-circuit cooling towers, closes-circuit cooling towers, and evaporative condensers used for comfort cooling applications must comply with Section C403.9

  – **Exception**: heat rejection devices where energy usage is included in the equipment efficiency ratings listed in Tables C403.3.2(6) and C403.3.2(7)
Fan motors > 5hp must have

- Controls that automatically **modulate** fan speed to control the leaving fluid temp. or condensing temp./pressure of the heat rejection device
- Fan motor power input to be not more than 30% of design wattage or 50% of design airflow

**Exception:**
- Condenser fans serving multiple refrigerant or fluid cooling circuits
- Condenser fans serving flooded condensers
Multiple-cell heat rejection equipment with variable speed fan drives to be controlled in both manners:

- Operate the maximum number of fans allowed that comply with manufacturer’s requirements for all system components
- So all fans can operate at the same fan speed required for the instantaneous cooling duty, as opposed to staged (on/off) operation

Minimum fan speed must be the minimum allowable speed of the fan drive system in accordance with manufacturer’s recommendations
Centrifugal fan open-circuit cooling towers with combined rated capacity > 1,100 gpm at 95°F condenser water return, 85°F condenser water supply, 75°F outdoor air wet-bulb temperature must meet the energy efficiency requirement for axial fan open-circuit cooling towers listed in Table C403.3.2(8)

**Exception:**
Centrifugal open-circuit cooling towers that are designed with inlet or discharge ducts or require external sound attenuation
Open-circuit cooling towers used on water-cooled chiller system that are configured with multiple or variable-speed condenser water pumps

- Designed that all open circuit cooling tower cells can be run in parallel with the larger of
  - the flow that is produced by the smallest pump at its minimum expected flow rate OR
  - 50% of the design flow for the cell
Condenser heat recovery required for heating/reheating of SWH provided:

- Facility operates 24 hours/day
- Total installed heat capacity of the heat rejection of water-cooled systems >6,000,000 Btu/hr
- Design SWH load >1,000,000 Btu/hr

Capacity to provide the smaller of

- 60% of peak heat rejection load at design conditions OR
- Preheating to raise peak SWH to 85ºF

Exceptions:

- Recovered heat is used for space heating or when 60% of SWH is heated by renewables or site recovered energy sources
Note: the following are generally in line with federal manufacturing requirements.

- Equipment have an energy use in kWh/day ≤ the values of Tables C403.10(1-2) when tested and rated in accordance with AHRI Standard 1200
- Energy use shall be verified through certification under an approved certification program or where a certification program does not exist, the energy use shall be supported by data furnished by the equipment manufacturer
Walk-in Coolers, Walk-in Freezers, Refrigerated Warehouse Coolers/Freezers (Mandatory)

Section C403.10.1

• Be equipped with automatic door-closers that firmly close walk-in doors that have been closed to within 1” of full closure
  – **Exception** – automatic closers are not required for doors >45” in width or > 7ft in height

• Doorways have strip doors, curtain, spring hinged doors or other approved method of minimizing infiltration when doors are open

• Walk-in coolers and refrigerated warehouse coolers shall have wall, ceiling, and door insulation of > R-25 and walk-in freezers and refrigerated warehouse freezers > R-32
  – **Exception** – glazed portions of doors or structural members need not be insulated
• Walk-in freezers contain floor insulation ≥ R-28
• Transparent reach-in doors for walk-in freezers and windows in walk-in freezer doors shall be of triple-pane glass, either filled with inert gas or with heat-reflective treated glass
• Windows and transparent reach-in doors for walk-in coolers, doors shall be of double-pane or triple pane, inert gas-filled, heat-reflective treated glass
• Evaporator fan motors that are < 1hp and < 460 volts use electronically commutated motors, brushless direct-current motors, or 3-phase motors
• Condenser fan motors < 1hp use electronically commutated motors, permanent split capacitor-type motors or 3-phase motors
• Antisweat heaters without antisweat heater controls:
  limit total door rail, glass and frame heater power draw < 7.1W/ft^2 of door opening for walk-in freezers and 3.0 W/ft^2 of door opening for walk-in coolers

• Where antisweat heater controls are provided
  reduce the energy use of the antisweat heater as a function of the relative humidity in the air outside the door or of the condensation on the inner glass plane
Site assembled or site constructed walk-ins have very similar provisions to pre-manufactured walk-ins covered under C403.10.1 with the following differences (underlined):

- **Exception to item 3:** insulation is not required for glazed portions of doors or at structural members associated with the walls, ceiling or door frame.

- **Exception to item 8:** Fan motors in *walk-in coolers* and *walk-in freezers* combined in a single enclosure greater than 3,000 square feet (279 m²) in floor area are exempt.

- **C403.10.2.1 Performance standards (Mandatory).** Effective January 1, 2020, *walk-in coolers* and *walk-in freezers* shall meet the requirements of Tables C403.10.2.1(1), C403.10.2.1(2) and C403.10.2.1(3).
Refrigerated Display Cases (Mandatory)

Section C403.10.3

Site-assembled or site-constructed refrigerated display cases shall comply with the following:

• Lighting and glass doors controlled by one of the following:
  – Time switch controls to turn off lights during nonbusiness hours. Timed overrides for display case shall turn the lights on for up to 1 hour and automatically time out to turn the lights off
  – Motion sensor controls on each display case section that reduce lighting power by at least 50% within 3 minutes after the area within the sensor range is vacated

• Low-temp. display cases incorporate temp.-based defrost termination control with a time-limit default. The defrost cycle shall terminate first on an upper temp. limit breach and second upon a time limit breach.

• Antisweat heater controls to reduce the energy use of the antisweat heater as a function of the relative humidity in the air outside the door or to the condensation on the inner glass pane
Display cases, walk-in coolers or walk-in freezers served by remote compressors and remote condensers not located in a condensing unit must comply with Sections C403.10.4.1 and C403.10.4.2.

Exception: Systems where the working fluid in the refrigeration cycle goes through both subcritical and supercritical states (transcritical) or that use ammonia refrigerant.
Fan-Powered Condensers

- Design saturated condensing temperatures for air-cooled condenser not to exceed
  - design dry-bulb temp. plus 10°F for low-temp. refrigeration systems
  - Design dry-bulb temp. plus 15°F for medium temp. refrigeration systems where saturated condensing temp. for blend refrigerants should be determined using the average of liquid vapor temps. as converted from the condenser drain pressure
- Condenser fan motors < 1 hp use electronically commutated motors, permanent split-capacitor-type motors or 3-phase motors
Air-cooled condensers, evaporatively cooled condensers, air- or water-cooled fluid coolers or cooling towers must reduce fan motor demand ≤ 30% of design wattage at 50% of design air volume, and incorporate one of the following continuous variable speed fan control approaches:

- Control for air-cooled condensers must use variable setpoint control logic to reset the condensing temp. setpoint in response to ambient dry-bulb temp.
- Control for evaporatively cooled condensers must use variable setpoint control logic to reset the condensing temp. setpoint in response to ambient wet-bulb temp.

- Multiple fan condensers to be controlled in unison
- Minimum condensing temp. setpoint ≤ 70ºF
• Compressors and multiple-compressor system suction groups must include control systems that use floating suction pressure control logic to reset the target suction pressure temp. based on the temp. requirements of the attached refrigeration display cases or walk-ins:

**Exception**: controls are not required for:

- Single-compressor system that do not have variable capacity capability

- Suction groups that have:
  
  • a design saturated suction temp. of ≥ 30°F,
  
  • comprise the high stage of a two-stage or cascade system OR
  
  • primarily serve chillers for secondary cooling fluids
• Liquid subcooling must be provided for all low-temp. compressor systems with a design cooling capacity \( \geq 100,000 \text{ Btu/hr} \) with a design-saturated suction temp. \( \leq -10^\circ F \)
  
  – Sub-cooled liquid temp. to be controlled at max. temp. setpoint of 50\(^\circ\)F at the exit of the subcooler using either compressor economizer ports or a separate compressor suction group operating at a saturated suction temp. \( \geq 18^\circ F \)

• Insulation for liquid lines with a fluid operating temp. \( < 60^\circ F \) must comply with Table C403.11.3

• Compressors that incorporate internal or external crankcase heaters must provide a means to cycle the heaters off during compressor operation
Insulation required for supply and return ducts and plenums

- Located in unconditioned space:
  - minimum R-6
- Duct located outside the building; duct or plenum within building envelope assembly shall be separated from building exterior or unconditioned or exempt spaces:
  - minimum R-8, Climate Zones 1-4
  - minimum R-12, Climate Zones 5-8

**Exceptions:**

- When located within equipment
- When design temperature difference between interior and exterior of the duct or plenum doesn’t exceed 15°F
C403.11.2.1 Low Pressure: Ducts designed to operate at static pressures \( \leq 2 \) in. w.g.

Securely fastened and sealed

**Exception**: Locking-type longitudinal joints and seams, other than the snap-lock and button-lock types, need not be sealed as specified in this section.

C402.11.2.2 Medium Pressure: Ducts designed to operate at static pressures \( > 2 \) in. w.g. but \( < 3 \) in. w.g.

Insulated and sealed in accordance with Section C403.11.1
Ducts and plenums designed to operate at static pressures > 3 in. w.g. to be

- insulated and sealed
- be leak tested in accordance with *SMACNA HVAC Air Duct Leakage Test Manual with the following result*

  ✓ Air leakage rate (CL) ≤ 4.0
  ✓ CL = F/P^{0.65}

Where:

- F = leakage rate cfm per 100 sf of duct surface area
- P = the static pressure of the test

Must document test of ≥ 25% of the duct area and meet the requirements
All piping serving heating or cooling system must be insulated in accordance with Table C403.11.3

Minimum Pipe Insulation
(thickness in inches)

<table>
<thead>
<tr>
<th>FLUID OPERATING TEMPERATURE RANGE AND USAGE (°F)</th>
<th>INSULATION CONDUCTIVITY</th>
<th>NOMINAL PIPE OR TUBE SIZE (inches)</th>
</tr>
</thead>
<tbody>
<tr>
<td>Conductivity Btu • in./(h • ft² • °F)°</td>
<td>Mean Rating Temperature, °F</td>
<td>&lt; 1</td>
</tr>
<tr>
<td>&gt; 350</td>
<td>0.32 – 0.34</td>
<td>250</td>
</tr>
<tr>
<td>251 – 350</td>
<td>0.29 – 0.32</td>
<td>200</td>
</tr>
<tr>
<td>201 – 250</td>
<td>0.27 – 0.30</td>
<td>150</td>
</tr>
</tbody>
</table>

(Partial table)

11.3.1: Piping insulation exposed to weather shall be protected from sun or moisture damage
Exceptions to pipe insulation requirements:

- Piping internal to HVAC equipment (*including fan coil units*) factory installed and tested
- Piping for fluid in temperature range $60\,\text{°F} < \text{temp} < 105\,\text{°F}$
- Piping for fluid not heated or cooled by electricity or fossil fuels
- Strainers, control valves, and balancing valves associated with piping $\leq 1\,\text{”}$ in diameter
- Direct buried piping for fluids $\leq 60\,\text{°F}$
Systems are to be radiant systems

Controlled by an occupancy sensing device or timer switch

✓ So system is automatically deenergized when no occupants are present
Snow Melt Systems (Mandatory)
Section C403.12.2

Snow- and ice-melting systems, supplied through energy service to the building, shall include

- automatic controls configured to shut off the system when the pavement temperature is above 50°F (10°C) and no precipitation is falling
- an automatic or manual control configured to shut off when the outdoor temperature is above 40°F (4°C)
Systems such as heat tracing of outdoor piping and heat exchangers, including self-regulated heat tracing to include:

- Automatic controls configured to shut off the system when outdoor air temperatures are > 40°F (4°C) OR
- When conditions of the protected fluid will prevent freezing
Table C404.2 Minimum Performance of Water-Heating Equipment

- Water Heater Types Covered:
  - Electric Storage
  - Gas and Oil Storage
  - Instantaneous Water Heaters – Gas and Oil
  - Hot water boilers – gas and oil
  - Pool heaters
  - Unfired storage tanks

Heat Traps (C404.3)
Piping Insulation (C404.4)
Heated Water Supply Piping (C404.5)
Circulation & Temperature Maintenance (C404.6)
Demand Recirculation (C404.7)
Drain Heat Recovery (C404.8)
Pools and Spas (C404.9)
Portable Spas (C404.10)

All service water heating items are mandatory
• Water-heating equipment and hot water storage tanks must meet Table C404.2
• Efficiency verified through data furnished by manufacturer of equipment or through certification under an approved certification program
• Water-heating equipment intended to be used to provide space heating must meet Table C404.2
High Input-rated SWH Systems
(Mandatory)
Section C404.2.1

- Gas-fired equipment installed in new buildings
- Single piece serves entire building with input rating \( \geq 1,000,000 \text{ Btu/h} \)
  - Thermal efficiency \( \geq 90\% \)
- Multiple pieces with combined input rating \( \geq 1,000,000 \text{ Btu/h} \)
  - Combined input-capacity-weighted-average thermal efficiency \( \geq 90\% \)

Exceptions:
- 25% of annual SWH requirement is provided by on-site renewable energy or site-recovered energy
- Input rating of water heaters installed in individual dwelling units
- Individual units with input rating \( \leq 100,000 \text{ Btu/h} \) not considered part of building SWH equipment
Heat Traps (Mandatory)
Section C404.3

- Storage tank-type water heaters and hot water storage tanks that have vertical water pipes connecting to inlet and outlet to be provided integral heat traps at those inlets and outlets or have pipe-configured heat traps in piping connected to those inlets and outlets.
- Tank inlets and outlets associated with solar water heating system circulation loops not required to have heat traps.
Insulation of Piping (Mandatory)
Section C404.4

- Piping from water heater to termination of heated water fixture supply pipe (all recirculation piping) to be insulated per Table C403.11.3.
- Both inlet and outlet piping of storage water heater or heated water storage tank
  - Piping to a heat trap or first 8 ft. of piping, whichever is less.
- Piping that is heat traced per Table C403.11.3 or heat trace manufacturer instructions.
- Tubular piping insulation installed in accordance with insulation manufacturer’s instructions.
- Insulation to be continuous
  - Except where piping passes through a framing member.
- Minimum insulation thickness not to supersede any greater insulation thickness requirements necessary for the protection of piping from freezing temps. or personnel against external surface temps. on the insulation.
Exceptions to SHW piping insulation:

- Tubing from the connection at the termination of the fixture supply piping to a plumbing fixture or plumbing appliance
- Valves, pumps, strainers and threaded unions in piping that is ≤1” in nominal diameter
- Piping from user-controlled shower and bath mixing valves to the water outlets
- Cold-water piping of demand recirculation water system
- Tubing form hot drinking-water heating unit to the water outlet
- Piping at locations where a vertical support of the piping is installed
- Piping surrounded by building insulation with a thermal resistance ≥ R-3
For piping from the nearest source of heated water (from the water heater or from the recirculation or trace heated loop) to fixture requires either maximum pipe length (C404.5.1) or maximum pipe volume (C404.5.2) and has maximum flow rated by size

- Flow rate through ¼” piping should be ≤ 0.5 gpm
- Flow rate through 5/16” piping should be ≤ 1.0 gpm
- Flow rate through 3/8” piping should be ≤ 1.5 gpm

**Intent is to reduce wasting previously-heated water that has cooled in pipes that do not require insulation**
Maximum allowed piping length from nearest source of heated water to termination of the fixture supply pipe:

Where piping contains more than one size, the largest size of pipe within the piping shall be used for determining the max. allowable length of piping in Table C404.5.1

- Public lavatory faucet, use “Public Lavatory faucets” column in Table C404.5.1
- All other plumbing fixtures and plumbing appliances use “Other fixtures and appliances” column in Table C404.5.1
• Water heaters, circulating water systems, and heat trace temp. maintenance systems to be considered sources of heated water

• Volume from the nearest source of heated water to the termination of the fixture supply pipe as follows:
  – Public lavatory facet: ≤ 2 ounces
  – Other plumbing fixtures or plumbing appliances: ≤ 0.5 gallon
• Volume to be the sum of the internal volumes of pipe, fittings, valves, meters and manifolds between the nearest source of heated water and the termination of the fixture supply pipe
• Volume determined from the “Volume” column in Table
• Volume contained with fixture shutoff valves, within flexible water supply connectors to a fixture fitting and within a fixture fitting should not be included in the water volume determination
• Heated water supplied by recirculating system or heat-traced piping, the volume should include the portion of the fitting on the branch pipe that supplies water to the fixture
• Circulation Systems
  – Controlled pump(s) required
  – Demand control required (see C404.7)
  – Gravity and thermosyphon not allowed

• Heat Trace Systems
  – Energy input adjusted to maintain temperature
  – Timed or demand automatic controls

• Controls for Hot Water Storage Tank Pumps
  – Automatic controls limit pump operation to no more than 5 minutes after heater operation

• Automatic controls, temperature sensors and pumps shall be in a location with access.
• Manual controls shall be in a location with ready access
• Systems with $\geq 1$ recirculation pumps that pump water from a heated-water supply pipe back to the heated-water source through a cold-water supply pipe must be a demand recirculation water system

• **Demand recirculation water systems** to have controls that:
  – Start pump upon receiving a signal from the action of a user of a fixture or sensing the flow of hot or tempered water to a fixture fitting or appliance
  – Limit the temp. of water entering the cold-water piping (used as a recirculation return) to $\leq 104^\circ F$
• Must comply with CSA B55.2
• Potable water-side pressure loss < 10 psi at maximum design flow
• Group R occupancies, units must meet efficiency in accordance with CSA B55.1

Note that this provision does not require the use of drain water heat recovery units; it just specifies their performance if used
Heaters (C404.9.1)

- Readily accessible on-off switch, that is an integral part of heater, mounted on the exterior of heater or external to within 3 feet of heater in a location with ready access
- Switch should not change the setting of the heater thermostat
- Switches to be in addition to a circuit breaker for the power to the heater
- Natural gas or LPG fired pool heaters will not have continuously burning pilot lights

Time switches or other control method (C404.9.2)

- Automatic controls required to control heaters and pumps on a preset schedule
- **Exceptions:**
  - Where public health standards require 24 hour operation
  - Where pumps are required to operate solar and waste heat recovery pool heating systems

Note: heaters, pumps and motors with built-in timers meet this requirement
Outdoor heated pools and outdoor permanent spas required to have a cover

- Cover must be vapor-retardant
- Or other approved vapor-retardant means

**Exception:**

- Pools deriving > 75% of operating season (of not fewer than 3 calendar months) energy for heating from site-recovered such as from heat pump or on-site renewable energy system
Building operations and maintenance information

 ✓ Documents provided to owner and consist of
   ✓ Manufacturers’ information
   ✓ Specifications and recommendations
   ✓ Programming procedures and data points
   ✓ Narratives
   ✓ And other means of illustrating how the building, equipment and systems are intended to be installed, maintained and operated

 ✓ Required regular maintenance actions for equipment and systems to be clearly stated on a readily visible label
   ✓ Label to include title or publication number for the operation and maintenance manual for that particular model and type of product
Prior to passing final mechanical and plumbing inspection
- Registered design profession to provide evidence of commissioning and completion

Construction document notes to clearly indicate provisions for commissioning and completion requirements
- Permitted to refer to specifications

Copies of all documents to be provided to the owner and made available to code official upon request
These systems are exempt from commissioning requirements

- In buildings where total mechanical equipment capacity is < 480,000 Btu/h (40 tons) cooling capacity and < 600,000 Btu/h combined service water heating and space-heating capacity

- Included in Section C403.3 that serve individual dwelling units and sleeping units
✓ Developed by registered design professional or approved agency and include:
  ✓ Narrative description of activities to be accomplished during each phase of commissioning
    ✓ Including personnel who will do each activity
  ✓ Listing of specific equipment, appliances or systems to be tested and description of tests to be performed
  ✓ Functions to be tested, including, but not limited to calibrations and economizer controls
  ✓ Conditions under which test will be performed
    ✓ At a minimum, testing will affirm winter and summer design conditions and full outside air conditions
  ✓ Measurable criteria for performance
Systems Adjusting and Balancing
Section C408.2.2

- HVAC systems balanced per generally accepted engineering standards
- Air and water flow rates measured and adjusted to deliver final flow rates within tolerances in product specifications
- Test and balance activities to include air system and hydronic system balancing
Each supply air outlet and zone terminal device equipped with means for air balancing per Chapter 6 of the IMC

No discharge dampers used for air-system balancing on constant volume fans and variable volume fans with motors ≥ 10 hp

Air systems balanced in a manner to first minimize throttling losses, then, for fans with system power > 1hp, fan speed adjusted to meet design flow conditions

Exception – fans with motors ≤ 1hp are not required to have a means for air balancing
Individual hydronic heating and cooling coils equipped with means for balancing and measuring flow

Hydronic systems proportionately balanced in a manner to first minimize throttling losses, then pump impeller to be trimmed or pump speed to be adjusted to meet design flow conditions

Each hydronic system to have either capability to measure pressure across the pump, or test ports at each side of each pump

Exceptions with a means for balancing or measuring flow:

- Pumps with pump motors ≤ 5hp
- Where throttling results in ≤ 5% of nameplate hp draw above that required if the impeller were trimmed
To demonstrate the installation and operation of components, systems, and system-to-system interfacing relationships in accordance with approved plans and specifications, such that operation, function, and maintenance serviceability for each of the commissioned systems is confirmed.

Testing to include all modes and sequence of operation, including under full-load, part-load and the following emergency conditions:
- All modes as described in the sequence of operation
- Redundant or automatic back-up mode
- Performance of alarms, and
- Mode of operation upon a lost off power and restoration of power

**Exception** – unitary or packaged HVAC equipment listed in Tables C403.2.3(1) through C403.2.3(3) that don’t require supply air economizers
HVAC and SWH control systems to be tested to document that control devices, components, equipment, and systems are calibrated, adjusted and operate in accordance with approved plans and specifications.

Sequences of operation to be functionally tested to document they operate in accordance with approved plans and specifications.

Air economizers to undergo a functional test to determine they operate in accordance with manufacturer’s specifications.
Registered design professional or approved agency to complete, certify, and submit to the building owner or owner’s authorized agent.

Report to be organized with mechanical and service hot water findings in separate sections to allow independent review. Report shall include Commissioning Compliance Checklist (Figure C408.2.4) and be identified as “Preliminary Commissioning Report” and will identify:

- Itemization of deficiencies found during testing that haven’t been corrected at the time of report preparation.
- Deferred tests that can’t be performed at the time of report preparation due to climatic conditions.
- Climatic conditions required for performance of deferred tests.
- Results of functional performance tests.
- Functional performance test procedures used during commissioning process, including measurable criteria for test acceptance.
Buildings or portions of buildings can’t pass final mechanical inspection until code official has received the Preliminary Commissioning Report from the building owner or owner’s authorized agent.
Completion of the following are required:

- Construction documents to specify that documents described in C408.2.5 be provided to building owner or owner’s authorized agent within 90 days of receipt of certificate of occupancy

  - C408.2.5.1 System balancing report per Section C408.2.2
    - descriptions of the activities and
    - measurements completed

  - C408.2.5.2 Final commissioning report with
    - Functional test results
    - Disposition of deficiencies found during testing
    - Functional performance test procedures

**Exception** – deferred tests which can’t be performed at time of report preparation due to climatic conditions